

What is the Proper Torque to Use on a Given Bolt?

“What torque should I use to tighten my bolts?” is a question suppliers of bolts are frequently asked by end user customers. Many times I have been asked if a chart is published on the recommended tightening torque for various bolt grades and sizes. I do not know of any. This article provides such a chart for “Initial Target Tightening Torque.” See Figure 1. The formula for generating these values is explained below.

The widely recognized engineering formula, $T = K \times D \times P$ (to be explained later in this article), was used to provide the chart’s values, but it must be understood that every bolted joint is unique and the optimum tightening torque should be determined for each application by careful experimentation.

A properly tightened bolt is one that is stretched such that it acts like a very ridged spring pulling mating surfaces together. The rotation of a bolt (torque) at some point causes it to stretch (tension). Several factors affect how much tension occurs when a given amount of tightening torque is applied.

The first factor is the bolt’s diameter. It takes more force to tighten a 3/4-10 bolt than to tighten a 3/8-16 bolt because it is larger in diameter. The second factor is the bolt’s grade. It takes more force to stretch an SAE Grade 8 bolt than it does to stretch an SAE Grade 5 bolt because of the greater material strength. The third factor is the coefficient of friction, frequently referred to as the “nut factor.” The value of this factor indicates that harder, smoother, and/or slicker bolting surfaces, such as threads and bearing surfaces, require less rotational force (torque) to stretch (tension) a bolt than do softer, rougher, and stickier surfaces.

The basic formula $T = K \times D \times P$ stated earlier takes these factors into account and provides users with a starting point for establishing an initial target tightening torque.

- T = target tighten torque (the result of this formula is in inch pounds, dividing by 12 yields foot pounds)
- K = coefficient of friction (nut factor), always an estimation in this formula
- D = bolt’s nominal diameter in inches
- P = bolt’s desired tensile load in pounds (generally 75% of yield strength)

The reason all applications should be evaluated to determine the optimum tightening torque is that the K factor in this formula is always an estimate. The most commonly used bolting K factors are 0.20 for plain finished bolts, 0.22 for zinc plated bolts, and 0.10 for waxed or highly lubricated bolts.

The only way to properly determine the optimum tightening torque for a given application is to simulate the exact application. This should be done with a tension indicating device of some type on the bolt in the application. The bolt is tightened until the desired P (load) is indicated by the tension indicating device. The tightening torque required to achieve the desired tension is the actual tightening torque that should be used for that given application.

It is extremely important to realize that this tightening value is valid only so long as all of the aspects of the application remain constant. Bolt suppliers sometimes have customers say that their bolts are no good because they have started breaking while being installed. Thorough investigation commonly reveals that the customer has started lubricating the bolts to make assembly easier, but maintained to same torque as was used when the were plain finished.

The table in this article shows that by using this formula a 1/2-13 Grade 5 plain bolt should be tightened to 82 foot pounds, but the same bolt that is waxed only requires 41 foot pounds to tighten the same tension. A perfect 1/2-13 Grade 5 waxed bolt will break if it is tightened to 81 foot pounds because the K factor is drastically lower. The bolts are fine, but the application changed. Suppliers need to understand this and be able to educate their customers to resolve this common customer complaint about breaking bolts.

IFI Technical Bulletin ®	<h3>What is the Proper Torque to Use on a Given Bolt?</h3>	IFI Page 1 of 2
Published and issued by the Industrial Fasteners Institute of Independence, OH		Issued: Sept., 1994

INDUSTRIAL FASTENERS INSTITUTE

6363 Oak Tree Boulevard • Independence, Ohio 44131 USA, www.indfast.org

The chart is provided for quick reference by fastener suppliers and users for selecting an initial target tightening torque. This chart was derived by using the formula shown earlier. An example of the calculation is as follows:

Product: 3/4-10 Grade 5 zinc plated bolt
Formula: $T = K \times D \times P$

- $K = 0.22$ (zinc plated)
- $D = .750$ (3/4-10 nominal diameter)
- $P = 23,046$ pounds

Note: $P = 75\%$ of yield strength (92,000 PSI) times the tensile stress area of the thread (0.3340)
 $T = (0.22 \times .750 \times 23,046) / 12 = 317$ foot pounds

Hopefully the chart will help suppliers with an initial answer to the customer's question, "What torque should I use to tighten my bolts?" Keep in mind this is only an estimated value. It may provide satisfactory performance, but it also may not. Every application should be evaluated on its own to determine the optimum torque value for each application. Major bolt suppliers should have tension indicating equipment necessary to help their customers determine the appropriate tightening values for their specific applications. Keep in mind that if the lubricant on a bolt and nut combination is changed, the tightening torque value must be altered to achieve the desired amount of bolt tension.

Thread Size	Tensile Stress Area TSA	SAE Grade 2			SAE Grade 5			SAE Grade 8		
		75% Yield Strength (PSI) = 43000			75% Yield Strength (PSI) = 69000			75% Yield Strength (PSI) = 98000		
		Plain	Zinc Plated	Waxed	Plain	Zinc Plated	Waxed	Plain	Zinc Plated	Waxed
		Square Inches	Ft. Lb.	Ft. Lb.	Ft. Lb.	Ft. Lb.	Ft. Lb.	Ft. Lb.	Ft. Lb.	R. Lb.
1/4-20.	0.0318	6	6	3	9	10	5	13	14	6
1/4-28.	0.0364	7	7	3	10	12	5	15	16	7
5/16-18.	0.0524	12	13	6	19	21	9	27	29	13
5/16-24.	0.0580	13	14	6	21	23	10	30	33	15
3/8-16.	0.0775	21	23	10	33	37	17	47	52	24
3/8-24.	0.0878	24	26	12	38	42	19	54	59	27
7/16-14.	0.1063	33	37	17	53	59	27	76	83	38
7/16-24.	0.1187	37	41	19	60	66	30	85	93	42
1/2-13.	0.1419	51	56	25	82	90	41	116	127	58
1/2-20.	0.1599	57	63	29	92	101	46	131	144	65
9/16-12.	0.1820	73	81	37	118	129	59	167	184	84
9/16-18.	0.2030	82	90	41	131	144	66	186	205	93
5/8-11.	0.2260	101	111	51	162	179	81	231	254	115
5/8-14.	0.2560	115	126	57	184	202	92	261	287	131
3/4-10.	0.3340	180	197	90	288	317	144	409	450	205
3/4-16.	0.3730	200	221	100	322	354	161	457	503	228

Figure 1. Initial Target Tightening Torque. (Formula: $T = K \times D \times P$)

Disclaimer: All information provided in this document and/or presentation is based on data the Industrial Fasteners Institute believes to be reliable and accurate. Such information is intended for use by persons at their own discretion and risk. The information here in is based on industry standards and recognized practices. The Industrial Fasteners Institute and the representatives providing this material assume no responsibility for the content.

IFI Technical Bulletin ®	<h2>What is the Proper Torque to Use on a Given Bolt?</h2>	IFI Page 2 of 2
Published and issued by the Industrial Fasteners Institute of Independence, OH		Issued: Sept., 1994

All standards and specifications are advisory only. Their use by anyone is entirely voluntary. Reliance thereon for any purpose by anyone is at the sole risk of that person or the user of the product, and the IFI is not responsible for any loss, claim, or damage arising therefrom. In formulating and approving standards and specifications, the IFI has not investigated and will not investigate patents which may apply to the subject matter. Prospective users of the standards and specifications are responsible for advising themselves of and protecting themselves against any patent infringement liability which may arise out of such use. © INDUSTRIAL FASTENERS INSTITUTE 2008